

A Practical Guide to 'Free Energy' Devices

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US Patent 1,938,497 12th November 1932 Inventor: Charles N. Pogue

CARBURETTOR

This invention relates to a device for obtaining an intimate contact between a liquid in a vaporous state and a gas, and particularly to such a device which may serve as a carburettor for internal combustion engines.

Carburettors commonly used for supplying a combustible mixture of air and liquid fuel to internal combustion engines, comprise a bowl in which a supply of the fuel is maintained in the liquid phase and a fuel jet which extends from the liquid fuel into a passage through which air is drawn by the suction of the engine cylinders. On the suction, or intake stroke of the cylinders, air is drawn over and around the fuel jet and a charge of liquid fuel is drawn in, broken up and partially vaporised during its passage to the engine cylinders. However, I have found that in such carburettors, a relatively large amount of the atomised liquid fuel is not vaporised and enters the engine cylinder in the form of microscopic droplets. When such a charge is ignited in the engine cylinder, only that portion of the liquid fuel which has been converted into the vaporous (molecular) state, combines with the air to give an explosive mixture. The remaining portion of the liquid fuel which is drawn into the engine cylinders and remains in the form of small droplets, does not explode and impart power to the engine, but burns with a flame and raises the temperature of the engine above that at which the engine operates most efficiently, i.e. 160⁰ to 180⁰ F.

According to this invention, a carburettor for internal combustion engines is provided in which substantially all of the liquid fuel entering the engine cylinder will be in the vapour phase and consequently, capable of combining with the air to form a mixture which will explode and impart a maximum amount of power to the engine, and which will not burn and unduly raise the temperature of the engine.

A mixture of air and liquid fuel in truly vapour phase in the engine cylinder is obtained by vaporising all, or a large portion of the liquid fuel before it is introduced into the intake manifold of the engine. This is preferably done in a vaporising chamber, and the "dry" vaporous fuel is drawn from the top of this chamber into the intake manifold on the intake or suction stroke of the engine. The term "dry" used here refers to the fuel in the vaporous phase which is at least substantially free from droplets of the fuel in the liquid phase, which on ignition would burn rather than explode.

More particularly, the invention comprises a carburettor embodying a vaporising chamber in the bottom of which, a constant body of liquid fuel is maintained, and in the top of which there is always maintained a supply of "dry" vaporised fuel, ready for admission into the intake manifold of the engine. The supply of vaporised liquid fuel is maintained by drawing air through the supply of liquid fuel in the bottom of the vaporising chamber, and by constantly atomising a portion of the liquid fuel so that it may more readily pass into the vapour phase. This is preferably accomplished by a double-acting suction pump operated from the intake manifold, which forces a mixture of the liquid fuel and air against a plate located within the chamber. To obtain a more complete vaporisation of the liquid fuel, the vaporising chamber and the incoming air are preferably heated by the exhaust gasses from the engine. The carburettor also includes means for initially supplying a mixture of air and vaporised fuel so that starting the engine will not be dependent on the existence of a supply of fuel vapours in the vaporising chamber.

The invention will be further described in connection with the accompanying drawings, but this further disclosure and description is to be taken as an exemplification of the invention and the same is not limited thereby except as is pointed out in the claims.

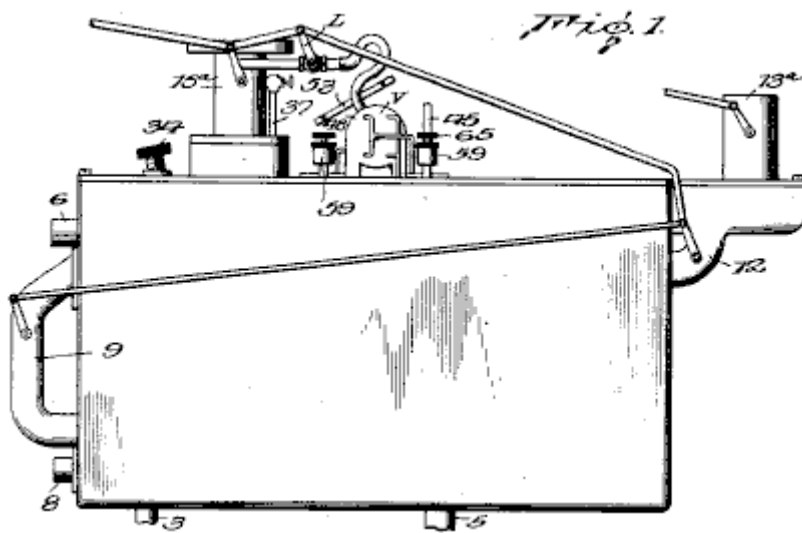


Fig.1 is an elevational view of a carburettor embodying my invention.

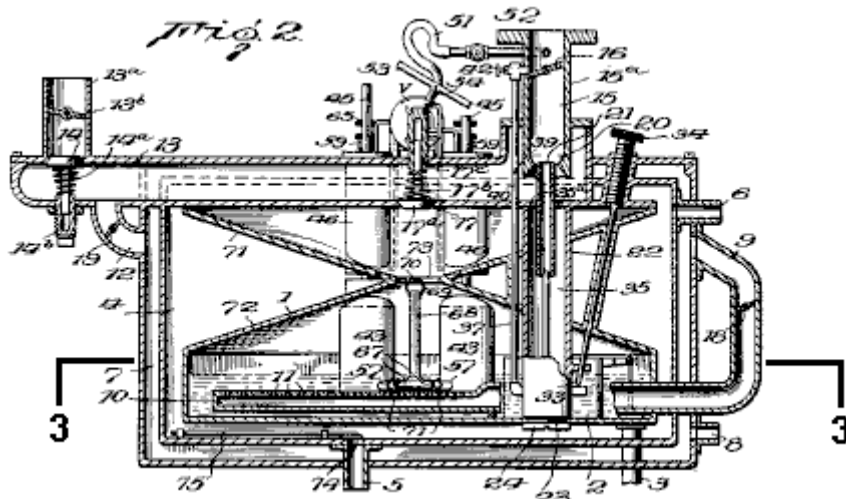


Fig.2 is a vertical cross-sectional view through the centre of Fig.1

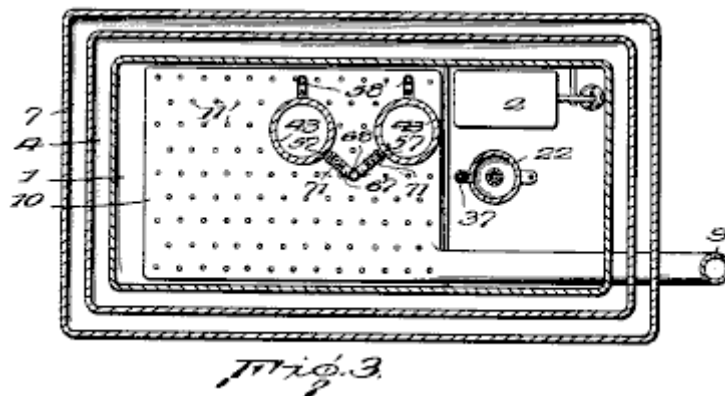


Fig.3 is a horizontal sectional view on line 3-3 of Fig.2.

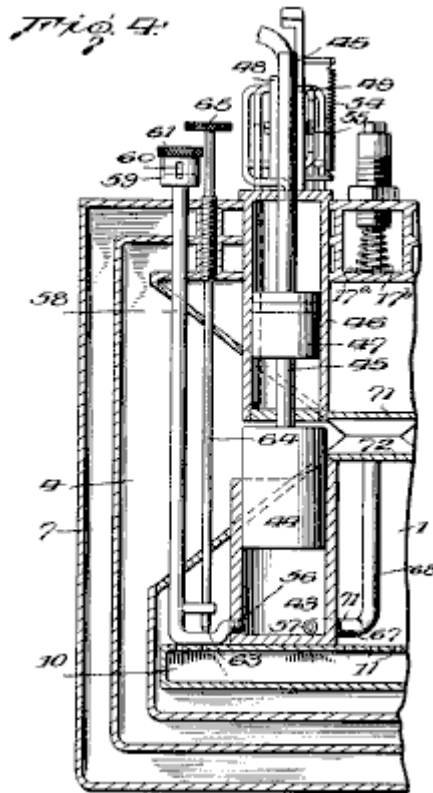


Fig.4 is an enlarged vertical sectional view through one of the pump cylinders and adjacent parts of the carburettor.

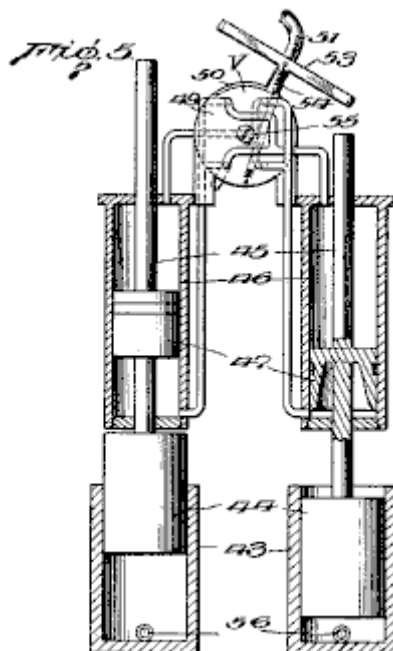


Fig.5 is an enlarged view through the complete double-acting pump and showing the associated distributing valve.

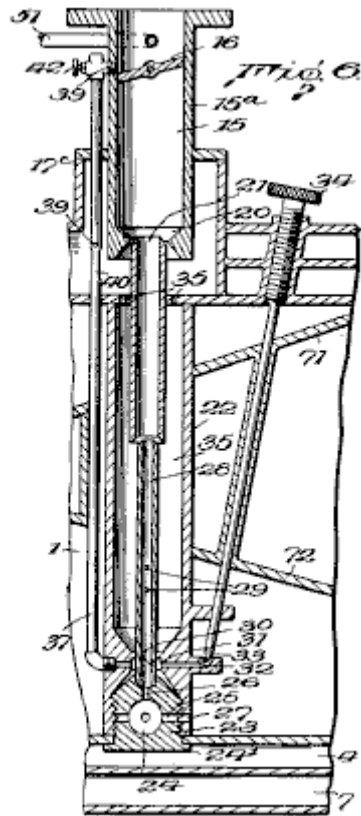


Fig.6 is an enlarged vertical sectional view through the atomising nozzle for supplying a starting charge for the engine.

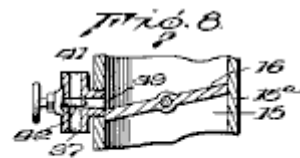
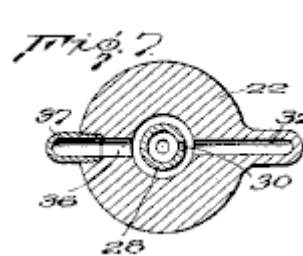
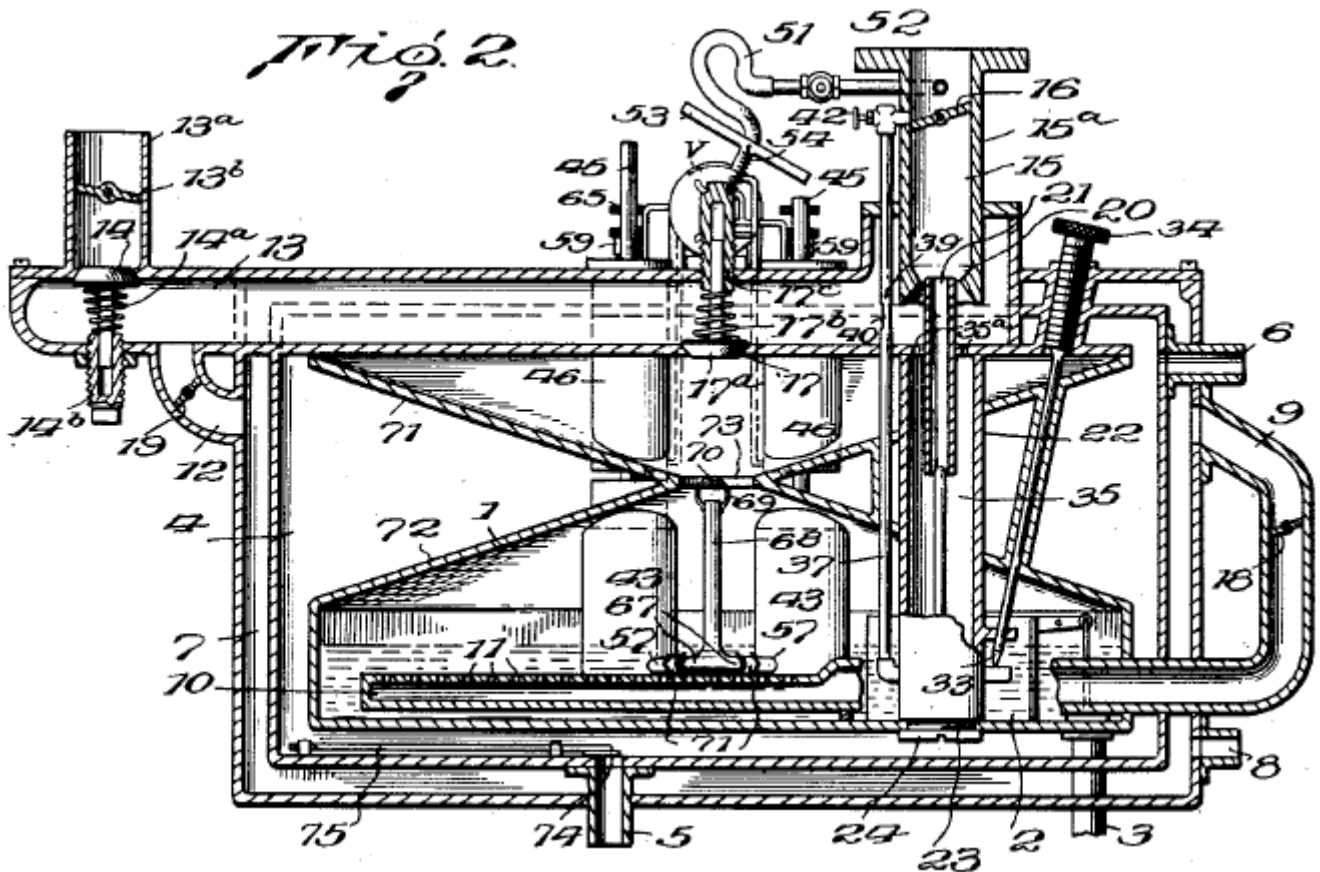


Fig.7 and **Fig.8** are detail sectional views of parts 16 and 22 of Fig.6



Fig.9 and **Fig.10** are detail sectional views showing the inlet and outlet to the cylinders of the atomising pump.



Referring to the drawings, the numeral 1 indicates a combined vaporising chamber and fuel bowl in which liquid fuel is maintained at the level indicated in Fig. 1 by a float-valve 2 controlling the flow of liquid fuel through pipe 3 which leads from the vacuum tank or other liquid fuel reservoir.

The vaporising chamber 1 is surrounded by a chamber 4 through which hot exhaust gasses from the engine, enter through pipe 5 located at the bottom of the chamber. These gasses pass around the vaporising chamber 1 and heat the chamber, which accelerates the vaporisation of the liquid fuel. The gasses then pass out through the upper outlet pipe 6.

Chamber 4 for the hot exhaust gasses, is in turn surrounded by chamber 7 into which air for vaporising part of the liquid fuel in chamber 1 enters through a lower intake pipe 8. This air passes upwards through chamber 4 through which the hot exhaust gasses pass, and so the air becomes heated. A portion of the heated air then passes through pipe 9 into an aerator 10, located in the bottom of the vaporising chamber 1 and submerged in the liquid fuel in it. The aerator 10 is comprised of a relatively flat chamber which extends over a substantial portion of the bottom of the chamber and has a large number of small orifices 11 in its upper wall. The heated air entering the aerator passes through the orifices 11 as small bubbles which then pass upwards through the liquid fuel. These bubbles, together with the heat imparted to the vaporising chamber by the hot exhaust gasses, cause a vaporisation of a portion of the liquid fuel.

Another portion of the air from chamber 7 passes through a connection 12 into passage 13, through which air is drawn directly from the atmosphere into the intake manifold. Passage 13 is provided with a valve 14 which is normally held closed by spring 14a, the tension of which may be adjusted by means of the threaded plug 14b. Passage 13 has an upward extension 13a, in which is located a choke valve 13b for assisting in starting the engine. Passage 13 passes through the vaporising chamber 1 and has its inner end communicating with passage 15 via connector 15a which is secured to the intake manifold of the engine. Passage 15 is provided with the usual butterfly valve 16 which controls the amount of fuel admitted to the engine cylinders, and consequently, regulates the speed of the engine.

The portion of passage 13 which passes through the vaporising chamber has an opening 17 normally closed by valve 17a which is held against its seat by spring 17b, the tension of which may be adjusted by a threaded plug 17c. As air is drawn past valve 14 and through passage 13 on the intake or suction stroke of the engine, valve 17a will be lifted from its seat and a portion of the dry fuel vapour from the upper portion of the vaporising chamber will be sucked into passage 13 through opening 17 and mingle with the air in it before entering passage 15.

In order to regulate the amount of air passing from chamber **7** to aerator **10** and into passage **13**, pipe **9** and connection **12** are provided with suitable valves **18** and **19** respectively. Valve **18** in pipe **9** is synchronised with butterfly valve **16** in passage **15**. Valve **19** is adjustable and preferably synchronised with butterfly valve **16** as shown, but this is not essential.

The bottom of passage **15** is made in the form of a venturi **20** and a nozzle **21** for atomised liquid fuel and air is located at or adjacent to the point of greatest restriction. Nozzle **21** is preferably supplied with fuel from the supply of liquid fuel in the bottom of the vaporising chamber, and to that end, a member **22** is secured within the vaporising chamber by a removable threaded plug **23** having a flanged lower end **24**. Plug **22** extends through an opening in the bottom of chamber **1**, and is threaded into the bottom of member **22**. This causes the bottom wall of chamber **1** to be securely clamped between the lower end of member **22** and flange **24**, thus securely retaining member **22** in place.

Plug **23** is provided with a sediment bowl **24** and extending from bowl **24** are several small passages **25** extending laterally, and a central vertical passage **26**. The lateral passages **25** register with corresponding passages **27** located in the lower end of member **22** at a level lower than that at which fuel stands in chamber **1**, whereby liquid fuel is free to pass into bowl **24**.

Vertical passage **26** communicates with a vertical nozzle **28** which terminates within the flaring lower end of nozzle **21**. The external diameter of nozzle **26** is less than the interior diameter of the nozzle **21** so that a space is provided between them for the passage of air or and vapour mixtures. Nozzle **26** is also provided with a series of inlets **29**, for air or air and vapour mixtures, and a fuel inlet **30**. Fuel inlet **30** communicates with a chamber **31** located in the member **22** and surrounding the nozzle **28**. Chamber **30** is supplied with liquid fuel by means of a passage **32** which is controlled by a needle valve **33**, the stem of which, extends to the outside of the carburettor and is provided with a knurled nut **34** for adjusting purposes.

The upper end of member **22** is made hollow to provide a space **35** surrounding the nozzles **21** and **28**. The lower wall of the passage **13** is provided with a series of openings **35a**, to allow vapours to enter space **35** through them. The vapours may then pass through inlets **29** into the nozzle **28**, and around the upper end of the nozzle **28** into the lower end of nozzle **21**.

Extending from chamber **31** at the side opposite passage **32**, is a passage **36** which communicates with a conduit **37** which extends upwards through passage **13**, and connects through a lateral extension **39**, with passage **15** just above the butterfly valve **16**. The portion of conduit **37** which extends through passage **13** is provided with an orifice **39** through which air or air and fuel vapour may be drawn into the conduit **37** mingle with and atomise the liquid fuel being drawn through the conduit. To further assist in this atomisation of the liquid fuel passing through conduit **37**, the conduit is restricted at **40** just below orifice **39**.

The upper end of conduit **37** is in communication with the atmosphere through opening **41** through which air may be drawn directly into the upper portion of the conduit. The proportion of air to combustible vapours coming through conduit **37** is controlled by needle valve **42**.

As nozzle **21** enters directly into the lower end of passage **15**, suction in the inlet manifold will, in turn, create a suction on nozzle **21** which will cause a mixture of atomised fuel and air to be drawn directly into the intake manifold. This is found to be desirable when starting the engine, particularly in cold weather, when there might not be an adequate supply of vapour in the vaporising chamber, or the mixture of air and vapour passing through passage **13** might be to "lean" to cause a prompt starting of the engine. At such times, closing the choke valve **13b** will cause the maximum suction to be exerted on nozzle **21** and the maximum amount of air and atomised fuel to be drawn directly into the intake manifold. After the engine has been started, only a small portion of the combustible air and vapour mixture necessary for proper operation of the engine is drawn through nozzle **21** as the choke valve will then be open to a greater extent and substantially all of the air and vapour mixture necessary for operation of the engine will be drawn through the lower end **20** of passage **15**, around nozzle **21**.

Conduit **37** extending from fuel chamber **31** to a point above butterfly valve **16** provides an adequate supply of fuel when the engine is idling with valve **16** closed or nearly closed.

The casings forming chambers **1**, **4** and **7**, will be provided with the necessary openings, to subsequently be closed, so that the various parts may be assembled, and subsequently adjusted or repaired.

The intake stroke of the engine creates a suction in the intake manifold, which in turn causes air to be drawn past spring valve **14** into passage **13** and simultaneously a portion of the dry fuel vapour from the top of vaporising chamber **1** is drawn through opening **17** past valve **17a** to mix with the air moving through the passage. This mixture then passes through passage **15** to the intake manifold and engine cylinders.

The drawing of the dry fuel vapour into passage **13** creates a partial vacuum in chamber **1** which causes air to be drawn into chamber **7** around heated chamber **4** from where it passes through connection **12** and valve **19**, into passage **13** and through pipe **9** and valve **18** into aerator **10**, from which it bubbles up through the liquid fuel in the bottom of chamber **1** to vaporise more liquid fuel.

To assist in maintaining a supply of dry fuel vapour in the upper portion of vaporising chamber **1**, the carburettor is provided with means for atomising a portion of the liquid fuel in vaporising chamber **1**. This atomising means preferably is comprised of a double-acting pump which is operated by the suction existing in the intake manifold of the engine.

The double-acting pump is comprised of a pair of cylinders **43** which have their lower ends located in the vaporising chamber **1**, and each of which has a reciprocating pump piston **44** mounted in it. Pistons **44** have rods **45** extending from their upper ends, passing through cylinders **46** and have pistons **47** mounted on them within the cylinders **46**.

Cylinders **46** are connected at each end to a distributing valve **V** which connects the cylinders alternately to the intake manifold so that the suction in the manifold will cause the two pistons **44** to operate as a double-acting suction pump.

The distributing valve **V** is comprised of a pair of discs **48** and **49** between which is located a hollow oscillatable chamber **50** which is constantly subjected to the suction existing in the intake manifold through connection **51** having a valve **52** in it. Chamber **50** has a pair of upper openings and a pair of lower openings. These openings are so arranged with respect to the conduits leading to the opposite ends of cylinders **46** that the suction of the engine simultaneously forces one piston **47** upwards while forcing the other one downwards.

The oscillatable chamber **50** has a T-shaped extension **53**. The arms of this extension are engaged alternately by the upper ends of the piston rods **45**, so as to cause valve **V** to connect cylinders **46** in sequence to the intake manifold.

Spring **54** causes a quick opening and closing of the ports leading to the cylinders **46** so that at no time will the suction of the engine be exerted on both of the pistons **47**. The tension between discs **48** and **49** and the oscillatable chamber **50** may be regulated by screw **55**.

The particular form of the distributing valve **V** is not claimed here so a further description of operation is not necessary. As far as the present invention is concerned, any form of means for imparting movement to pistons **47** may be substituted for the valve **V** and its associated parts.

The cylinders **43** are each provided with inlets and outlets **56** and **57**, each located below the fuel level in chamber **1**. The inlets **56** are connected to horizontally and upwardly extending conduits **58** which pass through the carburettor to the outside. The upper ends of these conduits are enlarged at **59** and are provided with a vertically extending slot **60**. The enlarged ends **59** are threaded on the inside to accept plugs **61**. The position of these plugs with respect to slots **60** determines the amount of air which may pass through the slots **60** and into cylinder **43** on the suction stroke of the pistons **44**.

The upper walls of the horizontal portions of conduits **58** have an opening **62** for the passage of liquid fuel from chamber **1**. The extent to which liquid fuel may pass through these openings is controlled by needle valves **63**, whose stems **64** pass up through and out of the carburettor and terminate in knurled adjusting nuts **65**.

The horizontal portion of each conduit **58** is also provided with a check valve **66** (shown in **Fig.10**) which allows air to be drawn into the cylinders through conduits **58** but prevents liquid fuel from being forced upwards through the conduits on the down stroke of pistons **44**.

Outlets **57** connect with horizontal pipes **67** which merge into a single open-ended pipe **68** which extends upwards. The upper open end of this pipe terminates about half way up the height of the vaporising chamber **1** and is provided with a bail **69** which carries a deflecting plate **70** positioned directly over the open end of pipe **68**.

The horizontal pipes **67** are provided with check valves **71** which permit the mingled air and fuel to be forced from cylinders **43** by the pistons **44**, but which prevent fuel vapour from being drawn from chamber **1** into cylinders **43**.

When operating, pistons **44** on the 'up' strokes, draw a charge of air and liquid fuel into cylinders **43**, and on the 'down' stroke, discharge the charge in an atomised condition through pipes **67** and **68**, against deflecting plate **70** which further atomises the particles of liquid fuel so that they will readily vaporise. Any portions of the liquid fuel which do not vaporise, drop down into the supply of liquid fuel in the bottom of the vaporising chamber where they

are subjected to the vaporising influence of the bubbles of heated air coming from the aerator **10**, and may again pass into the cylinders **43**.

As previously stated, the vaporised fuel for introduction into the intake manifold of the engine, is taken from the upper portion of the vaporising chamber **1**. To ensure that the vapour in this portion of the chamber shall contain no, or substantially no, entrained droplets of liquid fuel, chamber **1** is divided into upper and lower portions by the walls **71** and **72** which converge from all directions to form a central opening **73**. With the vaporising chamber thus divided into upper and lower portions which are connected only by the relatively small opening **73**, any droplets entrained by the bubbles rising from the aerator **10**, will come into contact with the sloping wall **72** and be deflected back into the main body of liquid fuel in the bottom of the chamber. Likewise, the droplets of atomised fuel being forced from the upper end of pipe **68** will, on striking plate **70**, be deflected back into the body of liquid fuel and not pass into the upper portion of the chamber.

In order that the speed of operation of the atomising pump may be governed by the speed at which the engine is running, and further, that the amount of air admitted from chamber **7** to the aerator **10**, and to passage **13** through connection **12**, may be increased as the speed of the engine increases, the valves **18**, **19** and **52** and butterfly valve **16** are all connected by a suitable linkage **L** so that as butterfly valve **16** is opened to increase the speed of the engine, valves **18**, **19** and **52** will also be opened.

As shown in **Fig.2**, the passage of the exhaust gasses from the engine to the heating chamber **4**, located between the vaporising chamber and the air chamber **7**, is controlled by valve **74**. The opening and closing of valve **74** is controlled by a thermostat in accordance with the temperature inside chamber **4**, by means of an adjustable metal rod **75** having a high coefficient of expansion, whereby the optimum temperature may be maintained in the vaporising chamber, irrespective of the surrounding temperature.

From the foregoing description, it will be understood that the present invention provides a carburettor for supplying to internal combustion engines, a comingled mixture of air and liquid fuel vapour free from microscopic droplets of liquid fuel which would burn rather than explode in the cylinders and that a supply of such dry vaporised fuel is constantly maintained in the carburettor.

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